MEC4047 Assignment 1

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Contents

[1. Heading 1 4](#_Toc506743677)

[1.1 Heading 2 4](#_Toc506743678)

# Analytical Solution

This section presents an analytical solution to the vibrating washing machine, including both steady-state and transient elements. The majority of calculations were done using MatLab R2022b, however these calculations and any algebraic derivations done have been transcribed in full in the following sections. Plotted results come from the MatLab implementation.

## Dynamic magnification ratio and phase angle

From the steady state free body diagram, the differential equation of motion for the rotating washing machine is as follows. The omega term is left as a symbolic variable as it can take on multiple values.

To find the dynamic magnification ratio and phase angle, the frequency ratio and the damping constant need to be calculated.

How do we calculate dynamic magnification ratio if we don’t have the frequency we are running the drum at?

Figure 1: Phase angle as function of frequency ratio

Chart, histogram

Description automatically generated

## Analytical solution for transient and steady-state displacement response

Three analytical solutions are presented for cases where the drum speed is equal to 400, 800 and 1200 rpm respectively. Before these values can be used in equations they need to be converted to rad/s:

The steady-state conditions for the washing machine drum and load are a typical case of rotational unbalance and can be found with standard formulae. First, the system values independent of the rotational speed of the drum are found:

The values above are independent of drum rotational speed. All following values differ depending on the drum speed used, and as such the formulae are presented and the final values are shown for each drum speed, but detailed substitutions are not shown for each case.

First the magnitude of the vertical response is calculated based on drum frequency:

Secondly, phase angle of the steady-state response can be found. First find the frequency ratio for the three drum speeds:

Then calculate steady state phase angle, psi:

Combining these two values (displacement and phase angle) shows the steady state displacement response of the washing machine. This is the particular solution to the differential equation of motion established in the first section.

Next is to calculate the transient response of the suspended drum as the rotation begins. Due to the presence of damping, no simple formula exists for the transient response. Derivation is needed to find the coefficients Aand B of the transient response.

Assume the complementary solution is of the form:

Then the full displacement of the spinning drum is as follows:

First substitute in the position at the start time, in this case 0:

Next find the derivative of y and substitute in the velocity at the start time, in this case also 0:

The constants A and B can be solved for any RPM value desired. This solving needs to occur for any RPM as changing the drum speed affects the magnitude and phase angle of the driving force. For 400, 800 and 1200 RPM, the values for A and B and C are shown in the table below. These can be used along with the formula for the full displacement response to plot the displacement response.

Table 1: Constants for displacement response for different drum speeds

|  |  |  |  |
| --- | --- | --- | --- |
| **RPM** | **A (m)** | **B (m)** | **C (m)** |
| 400 | 0.0269 | -0.0058 | 0.0273 |
| 800 | 0.0221 | -00032 | 0.0221 |
| 1200 | 0.0213 | -0.0029 | 0.0214 |

Plots of these graphs are now shown, as well as an estimate for how quickly the transient response dies away in natural periods for each case.

Figure 2: Washing machine displacement for 400 RPM.

A picture containing chart

Description automatically generated

Figure 3: Washing machine displacement for 800 RPM.

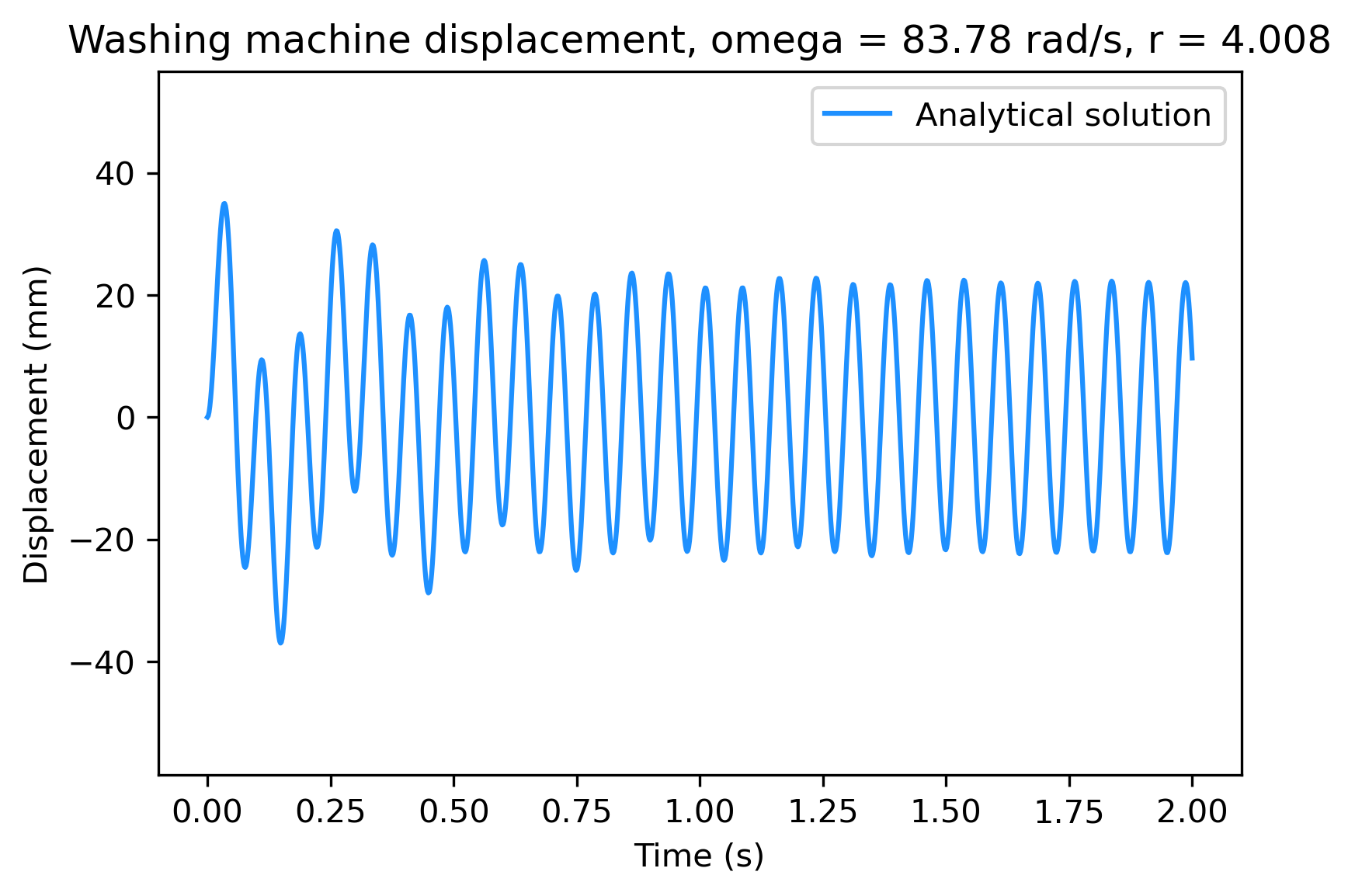
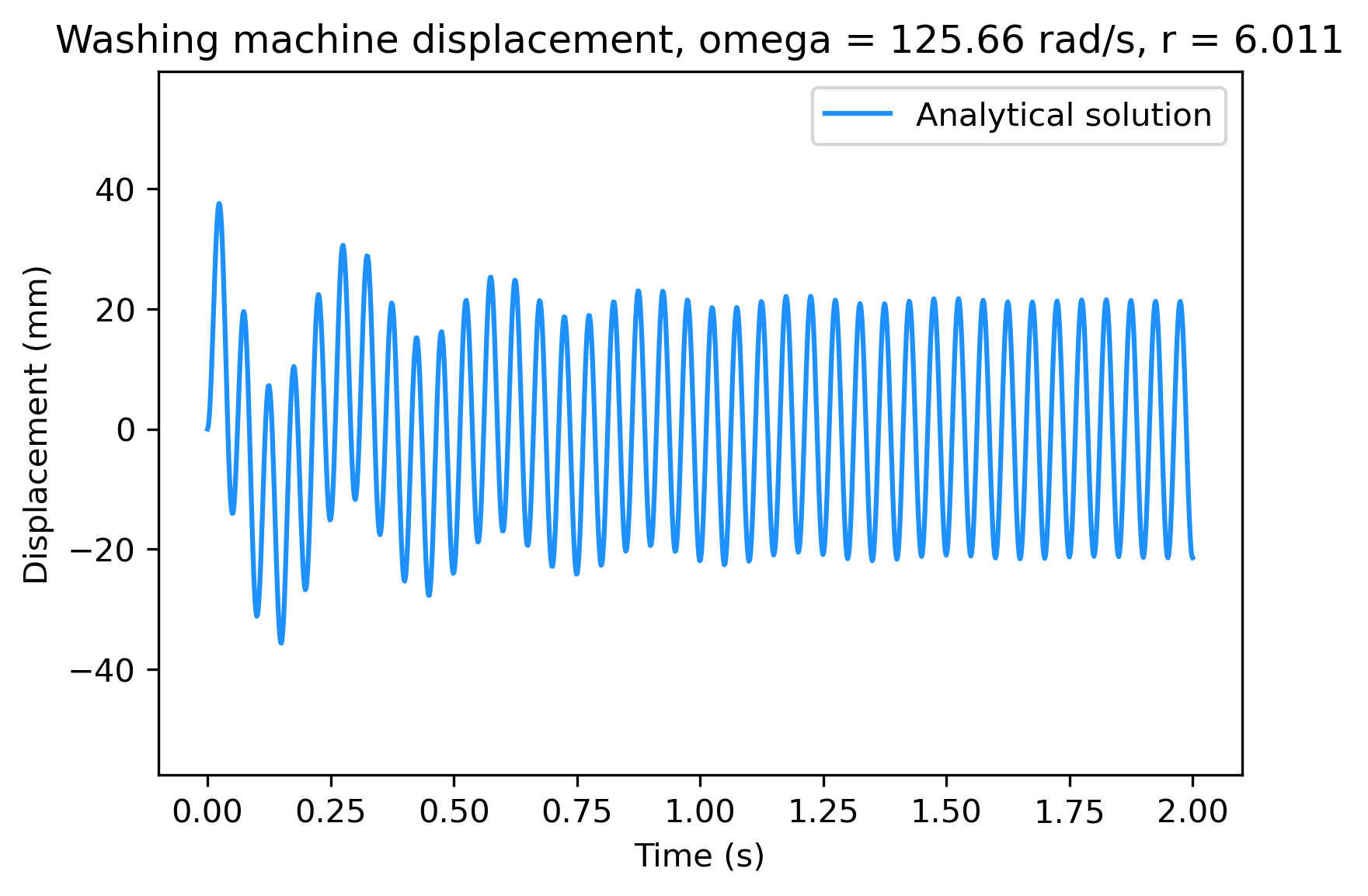


Figure 4: Washing machine displacement for 1200 RPM.



Counting natural periods until the transient response is no longer visible in the output displacement graphs. For 400 RPM it takes approximately 8 natural periods for the transient response to no longer by noticeable. For 800 RPM it takes approximately 17 natural periods and for 1200 RPM the number of natural frequencies increases to approximately 30 before the transient response is negligible.

## Force exerted on floor as function of frequency ratio.

An expression for force exerted on the floor needs to be derived from the displacement function for the washing machine. Due to the force being plotted against frequency ratio, the brief transient response of the washing machine is ignored for this analysis, and only steady state force is found as a function of frequency ratio.

Then the function for force on the floor is given by:

For rotational unbalance: